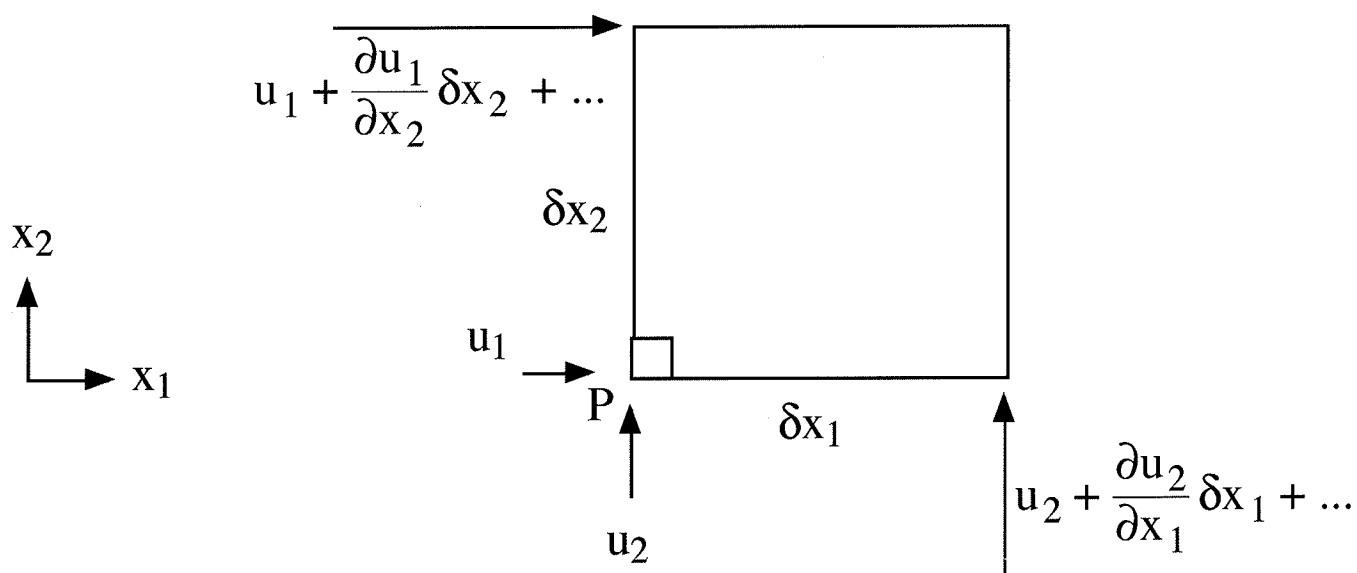


METR 5113, Advanced Atmospheric Dynamics I
 Alan Shapiro, Instructor
 Friday, 7 September 2018 (lecture 8)

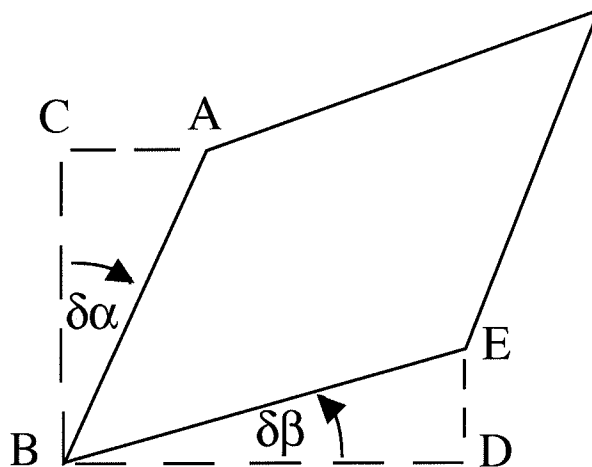
1 handout: Problem set 2

Shear strain rate: Rate of decrease of angle btw 2 mutually perpendicular lines in a fluid element.

Fluid element at time t :



Same fluid element at time $t + \delta t$:



So rate of shear strain is: $\lim_{\delta t \rightarrow 0} \frac{\delta\alpha + \delta\beta}{\delta t}$

Lets relate rate of shear strain to local flow conditions.

from geometry: $\tan\delta\alpha = \frac{x_A - x_C}{\delta x_2}$ [for small $\delta\alpha$: $\tan\delta\alpha \approx \delta\alpha$]

$$x_A = x_P + \left(u_1 + \frac{\partial u_1}{\partial x_2} \delta x_2 + \dots\right) \delta t + \text{h.o.t.}$$

$$x_C = x_B = x_P + u_1 \delta t + \text{h.o.t.}$$

$$\therefore \delta\alpha = \frac{\frac{\partial u_1}{\partial x_2} \delta x_2 \delta t + \text{h.o.t.}}{\delta x_2} = \frac{\partial u_1}{\partial x_2} \delta t + \text{h.o.t.}$$

Similarly can show that $\delta\beta = \frac{\partial u_2}{\partial x_1} \delta t + \text{h.o.t.}$

$$\text{So } \boxed{\lim_{\delta t \rightarrow 0} \frac{\delta\alpha + \delta\beta}{\delta t} = \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1}}$$

Define rate of strain tensor e by:

$$e_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

Note that: $e_{ij} = e_{ji}$ so e is a symmetric tensor.

Diagonal elements of e :

$$e_{11} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_1} + \frac{\partial u_1}{\partial x_1} \right) = \frac{\partial u_1}{\partial x_1} = \text{rate of normal strain in } x_1 \text{ dir}^n.$$

$$\text{Similarly, } e_{22} = \frac{\partial u_2}{\partial x_2}, \quad e_{33} = \frac{\partial u_3}{\partial x_3}.$$

\therefore Diagonal elements of e are normal strain rates.

\therefore Vol strain rate = sum of diag elements of e (trace of e)

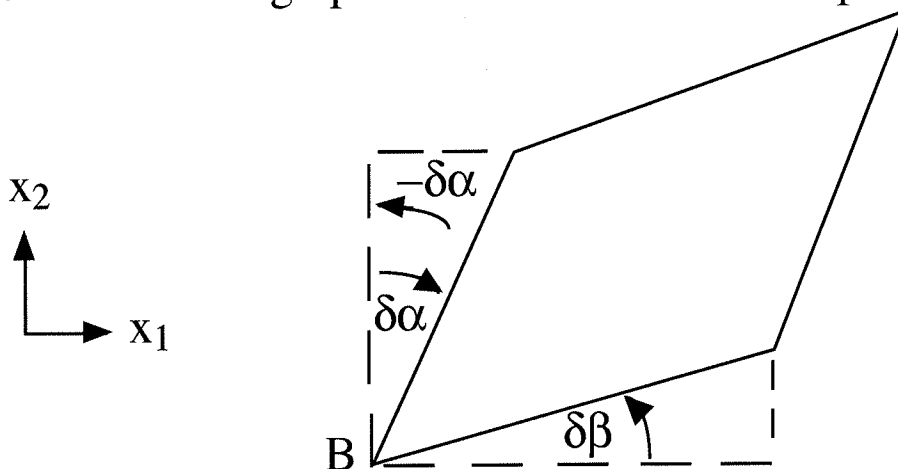
Off-diagonal elements of e :

$$e_{12} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \text{ rate of shear strain in } x_1 x_2 \text{ plane} .$$

\therefore Off-diagonal elements of e are half the rate of shear strains.

Vorticity

Calculate average angular velocity of 2 \perp lines in a fluid element about \perp axis through point of intersection. Use prev diagram:



"Average" or "local" angular velocity about x_3 axis through B is:

$$\Omega_3 = \lim_{\delta t \rightarrow 0} \frac{1}{2} \left(\frac{\delta\beta + (-\delta\alpha)}{\delta t} \right) = \frac{1}{2} \left(\frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right)$$

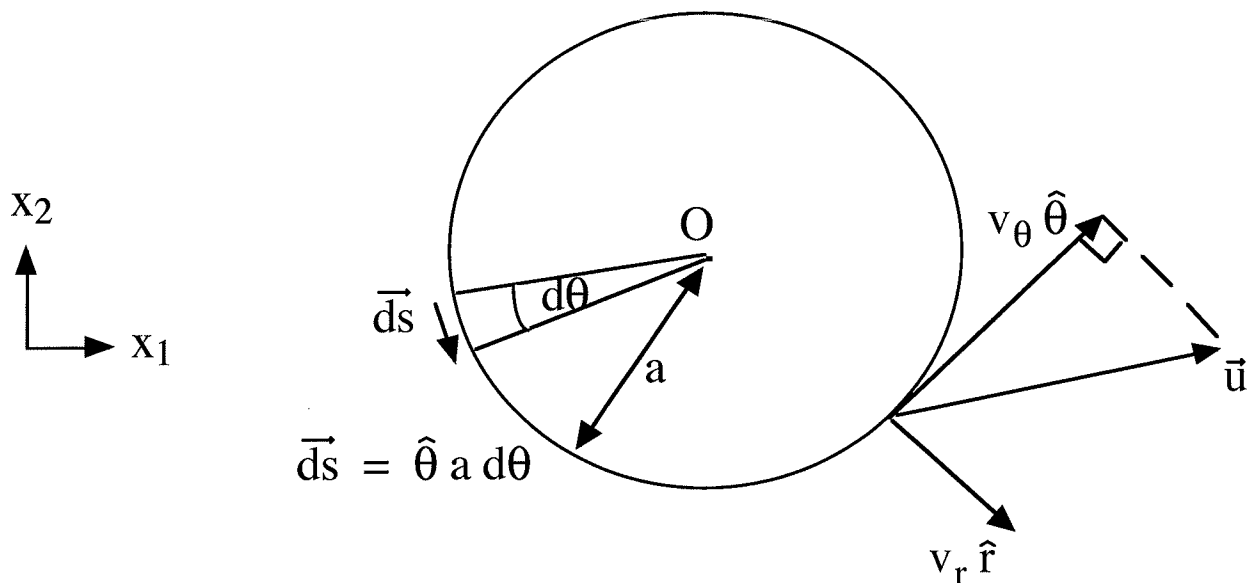
Define x_3 comp of vorticity to be twice this local ang velocity:

$$\omega_3 \equiv 2\Omega_3 = \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2}$$

Get similar formulas for local ang velocity about x_1 and x_2 axes.

$$\begin{aligned} \vec{\omega} &= \left(\frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3} \right) \hat{e}_1 + \left(\frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) \hat{e}_2 + \left(\frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) \hat{e}_3 \\ &= \varepsilon_{ijk} \frac{\partial u_k}{\partial x_j} \hat{e}_i = \nabla \times \vec{u} \end{aligned}$$

Another look at vorticity. Draw circle of small (vanishing) radius a around point of interest. Compute ave ang velocity on circle.



$\hat{\theta}$ is unit vector in tangential dirⁿ.

tangential velocity: $v_{\theta} = \vec{u} \cdot \hat{\theta}$

angular velocity: $\Omega_3 = \frac{v_{\theta}}{a}$

Calculate the average angular velocity around the circle:

$$\begin{aligned}\bar{\Omega}_3 &= \frac{1}{2\pi} \int_0^{2\pi} \Omega_3 d\theta = \frac{1}{2\pi} \int_0^{2\pi} \frac{v_{\theta}}{a} d\theta = \frac{1}{2\pi a} \int_0^{2\pi} \vec{u} \cdot \boxed{\hat{\theta} d\theta} = \vec{u} \cdot \vec{ds} / a \\ &= \frac{1}{\boxed{2\pi a^2}} \oint \vec{u} \cdot \vec{ds} = \frac{1}{2A} \oint \vec{u} \cdot \vec{ds} \quad \text{Now apply Stokes Thm} \\ &= \frac{1}{2A} \int (\nabla \times \vec{u}) \cdot \hat{n} dA, \quad \text{where } \hat{n} = \hat{e}_3\end{aligned}$$

For teeny-weeny circle (let $a \rightarrow 0$) the ave ang velocity is:

$$\begin{aligned}\bar{\Omega}_3 &= \lim_{a \rightarrow 0} \frac{1}{2} \frac{1}{A} \int (\nabla \times \vec{u}) \cdot \hat{e}_3 dA = \lim_{a \rightarrow 0} \frac{1}{2} \frac{1}{A} (\nabla \times \vec{u}) \cdot \hat{e}_3 A \\ &= \frac{(\nabla \times \vec{u}) \cdot \hat{e}_3}{2}\end{aligned}$$

$$\therefore \boxed{(\nabla \times \vec{u}) \cdot \hat{e}_3 = 2\bar{\Omega}_3}$$

define vertical vorticity: $\omega_3 \equiv (\nabla \times \vec{u}) \cdot \hat{e}_3$

So vorticity is twice local angular velocity.

It's important to understand the relevant length scales when discussing values of vorticity, e.g. tornado-like value of vorticity 1 s^{-1} can be easily generated by slamming a door. The 1 s^{-1} value is "large" if that value is maintained over an area that's say a few 100 m in radius.